#### NATIONAL TRANSPORTATION SAFETY BOARD

Office of Aviation Safety Aviation Engineering Division Washington, DC 20594

December 5, 2003

### ADDENDUM NUMBER 7 TO THE STRUCTURES GROUP CHAIRMAN'S FACTUAL REPORT

#### DCA02MA001

#### A. ACCIDENT

Location:	Belle Harbor, NY
Date:	November 12, 2001
Time:	09:16:14 EST
Aircraft:	American Airlines Flight 587, Airbus Model A300-605R, N14053
	Manufactures Serial Number (MSN) 420

### B. STRUCTURES GROUP

Chairman: Brian K Murphy National Transportation Safety Board Washington, DC

#### C. AIRBUS TEST REQUIREMENT

1. "Test program Rear Main Fitting A300-600R (Lug Test #1)"

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	Test Requirement							
Department:	ESGE	Repo	rt No.:	32	X 029 K4 804	P34		
Тур:	A300-600R							
Title:				Investigation itting A300-60	0R (Lug Test	<b>#1</b> )		
Summary:	CFRP fin box load condition	panel. T s to whice load co	he test s ch the fir	side rear main a shall demonstrate of AA flight 587 are derived fro	e the behavior o ' has been expo	f the lug under sed during the		
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# Test Requirement Rear Main Fitting

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# 1. Introduction

The test component is a LH-side rear main attachment lug from an A310 CFRP fin box panel. This panel originally represents a manufacturing test and has been used to demonstrate the interior quality. For this reason a large sample has been cut out of the panel above rib4 and between rear spar and stringer 7 (fig.9.1). The remaining part (fig 9.2) is prepared for clamping to the test rig by removal of the stringer run outs (webs and inboard flange) and reinforcement by additional plies (fig 9.3/4).

# 2. Test objective

The test shall demonstrate the behavior of the lug under tensile load condition to which the fin of AA flight 587 has been exposed during the accident. The load condition is derived from DFDR-data and structural analysis by FEM.

# 3. Test component

The removal of the stringer webs and inboard flanges and the reinforcing plies for bolting of the component is shown on the figure 9.4.

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## 4. Environmental conditions

The tests will be carried out at ambient temperature (RT). The component will be tested in an 'as received ' condition.

## 5. Test setup

The test specimen will be fixed to the test rig (fig 9.5/6) at the upper end (parallel to rib 4), at the rear spar, parallel to stringer 6 and at rib 1. The loads will be introduced in the global coordinate system by 3 servo-hydraulic-jacks. The lateral loads in Y-direction are introduced in line with the axis of the connecting pin.

The loads in X- and Z-direction are applied by two rods each which are adjustable in their length by turnbuckles (see chapter 9.6). Therefore the turnbuckles can be used to preadjust an initial local lug moment Mx and Mz, before applying the test load vector.

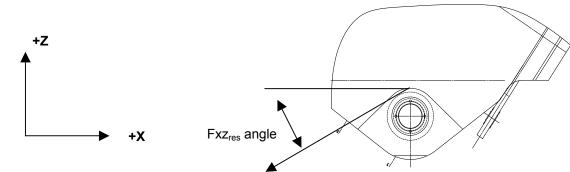
The original fin/fuselage connecting pin has to be used. The fitting is attached to a forkend lug (representing the fuselage attachment fitting) made from high strength aluminum alloy with similar dimensions as the original fuselage fitting for stiffness reason.

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## 6. Load case

In a meeting at Airbus Hamburg on the 12<sup>th</sup> of August 2003, it was agreed by NTSB, NASA, American Airlines and Airbus to select the NASA W375 MOD load vector for the Lug Test#1. This target condition includes a preadjusted local lug moment Mx of 2400Nm, which has to be introduced with the turnbuckles.

NASA W375	-Mod						
	Fx	Fy	Fz	Fres	Mx	Mz	Angle Xzplane
	[kN]	[kN]	[kN]	[kN]	[Nm]	[Nm]	[°]
W375-Mod	-400	-42	-864	953	6300	-1600	65
			•		•		



## 7. Measurement plan

Figure 9.7/8 shows the strain gauge positions and the orientation. Displacement measurement positions are shown in the fig. 9.9. The reference for the displacements is the support plate of the upper boundary of the test component.

# 8. Test procedure

# 8.1 Basic inspections

After reinforcement of the test specimen a visual and ultrasonic inspection has to be done to document the quality and condition.

A second inspection (visual and US) has to be carried out after installation into the test rig before starting the test.

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**8.2 Load case application** The forces are then applied in the following load steps:

	Load step	Fres	Fz	Fy	Fx
	0	400Nm	usted Mx of 2	uckles preadj	With turnbu
	10	48	-43	-2	-20
	15	71	-65	-3	-30
	20	95	-86	-4	-40
	25	119	-108	-5	-50
	30	143	-130	-6	-60
	35	162	-147	-7	-68
	40	186	-168	-8	-78
	45	210	-190	-9	-88
	50	233	-212	-10	-98
	55	257	-233	-11	-108
	60	281	-255	-12	-118
	65	305	-276	-13	-128
	70	329	-298	-14	-138
	75	353	-320	-16	-148
	80	376	-341	-17	-158
	85	400	-363	-18	-168
	90	424	-384	-19	-178
	95	448	-406	-20	-188
Limit Load	100	467	-423	-21	-196
level	105	491	-445	-22	-206
	110	515	-467	-23	-216
	115	538	-488	-24	-226
	120	562	-510	-25	-236
	125	586	-531	-26	-246
	130	610	-553	-27	-256
	135	634	-575	-28	-266
	140	658	-596	-29	-276
	145	681	-618	-30	-286
Ultimate Loa	150	705	-639	-31	-296
level	152	715	-648	-32	-300
	154	724	-657	-32	-304
	156	734	-665	-32	-308
	158	743	-674	-33	-312
	160 _	753	-683	-33	-316
NASA W375-Mo	203	953	-864	-42	-400
load vector	Load step	Fres	Fz	Fy	Fx

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Definition of load step:

Load step 100 is defined as 100% of Limit Load of the design gust case (BI17) with Fres=469kN.

From load step 160 (160% of Limit Load) the test is pursued by continuous proportional load increase up to the failure.

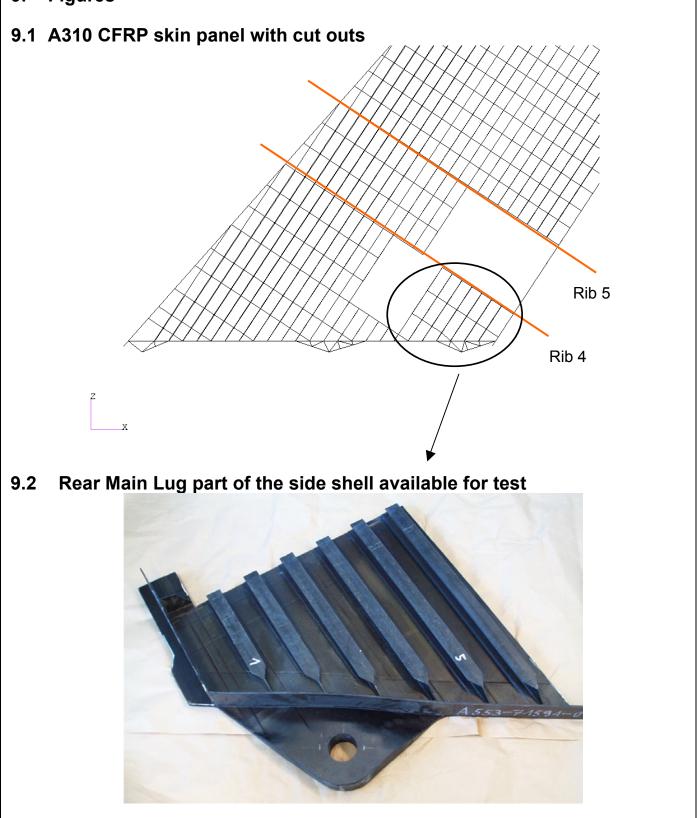
At each load step strain gauges, displacements and load cell values (see 7.) have to be recorded. After measuring the load step 160 all measurements are recorded continuously up to failure.

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# 9. Figures



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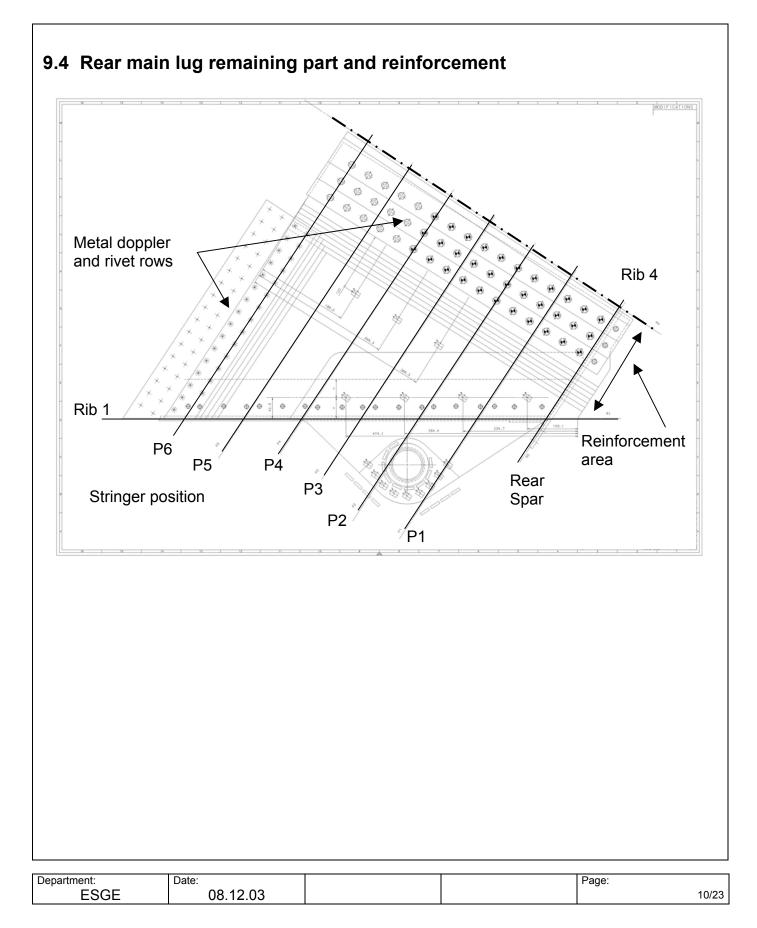
# 9.3 Reinforced test specimen with part of rib1 and the rear spar



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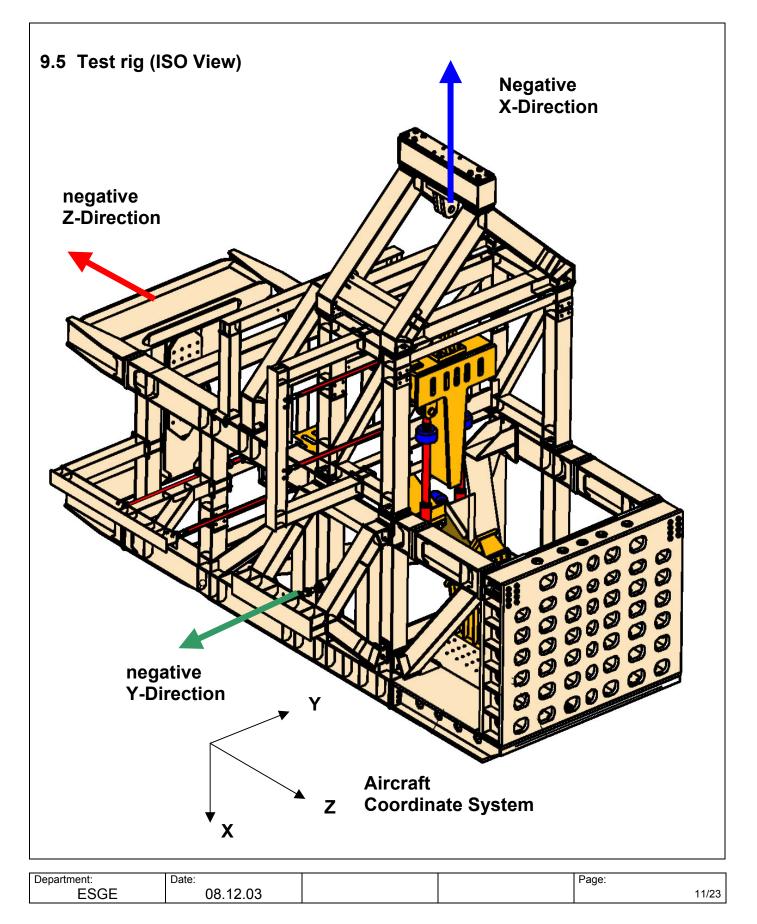


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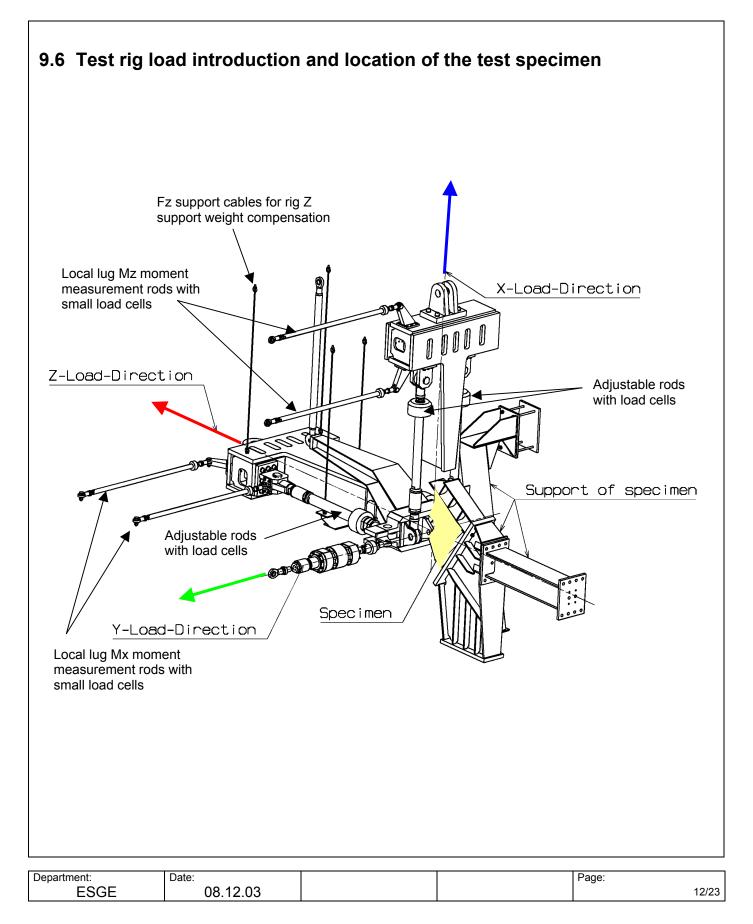




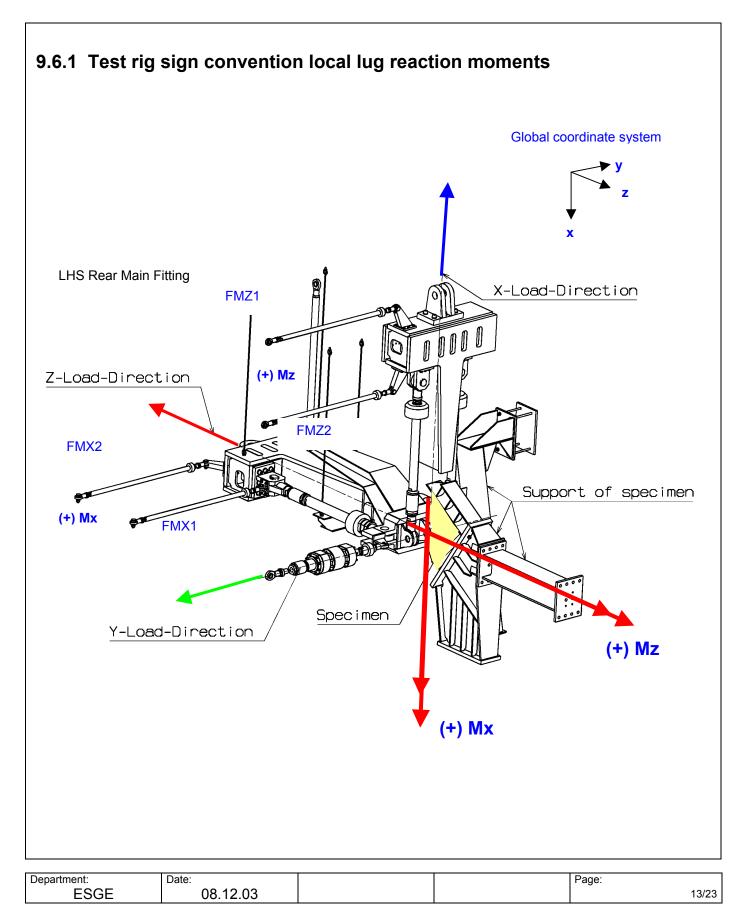
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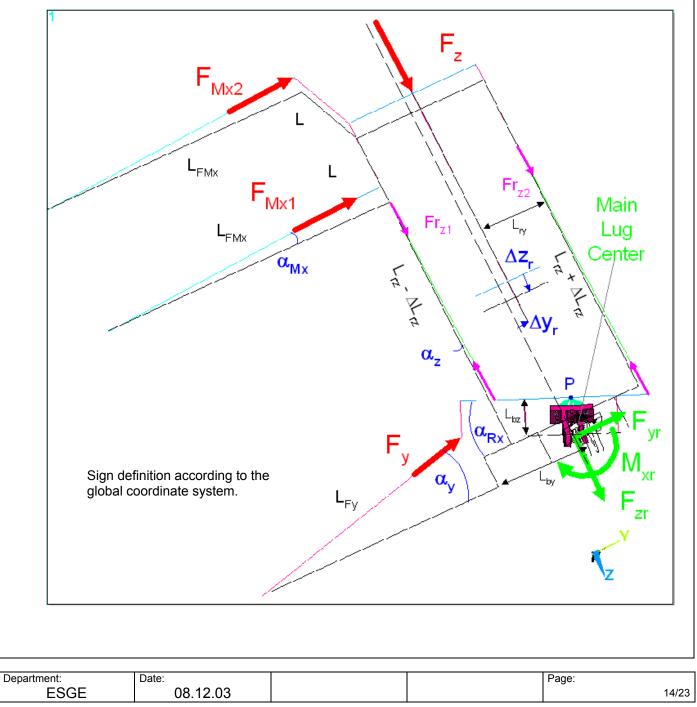






## 9.6.2 Local lug moment Mx (Equation considers displacements in the yz-plane)

The equation represents the moment equilibrium at the displaced load introduction system due to the deformed test specimen (see figure below). The supports of the Fy- actuator and the rods ends of FMX1/2 are assumed to be fixed and the free play at all bearings and length deformation of the linkages are not taken into account.



For practical reasons, the reference for the displacement measurement is the test rig. This can also cause some errors in the measured values.

The figure shows the undeformed and deformed load introduction and the relevant dimension in the test rig.

Test rig dimension:

Dimension			Description
L <sub>Fy</sub>	[mm]	1783	Y-cylinder length from outer to inner bearing point
L <sub>by</sub>	[mm]	720	Inner Y-cylinder bearing point to the lug reference point
L <sub>bz</sub>	[mm]	245	Z-distance from lug reference point to the z-axis main rod bearing points
L <sub>ry</sub>	[mm]	500	Half distance between the z-axis main rods Z1 and Z2
L <sub>rz</sub>	[mm]	2000	Length of the main rods Z1 and Z2
L <sub>FMx</sub>	[mm]	2000	Length of the moment measurement rods FMX1 and FMX2
L	[mm]	500	Half distance between FMX1 and FMX2

Equations to recalculate the local lug reactions with the measured data:

## Assumptions:

- 1. Deformation of the test rig + fuselage clevis negligible
- 2. Only displacement in the yz-plane considered

## **Displacement:**

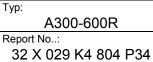
 $\Delta z P(\alpha Rx, \Delta zr) = \Delta zr + L_{bz}(1 - \cos(\alpha Rx))$  Z-Displacement in point P

 $\Delta y P(\alpha Rx, \Delta yr) = \Delta yr + L_{bz} \cdot \sin(\alpha Rx) \text{ Y-Displacement in point P}$ 

 $\Delta zsy(\alpha Rx, \Delta zr) = \Delta zr - L_{bv} \cdot \sin(\alpha Rx)$  Z-Displacement sum Fy

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**Deformation angle:** 

$$\alpha Rx = \arcsin\left(\frac{Dz3 - Dz4}{600}\right)$$
 Rx bolt rotation  
 $\alpha_y(\alpha Rx, \Delta zr) = \arcsin\left(\frac{\Delta zsy(\alpha Rx, \Delta zr)}{L_{Fy}}\right)$ 

$$\alpha_{z}(\alpha Rx, \Delta yr) = \arcsin\left(\frac{\Delta y P(\alpha Rx, \Delta yr) + L_{ry}(1 - \cos(\alpha Rx))}{L_{rz}}\right)$$
$$\alpha_{Mx}(\alpha Rx, \Delta zr) = \arcsin\left(\frac{\Delta z P(\alpha Rx, \Delta zr)}{L_{FMx}}\right)$$

#### Total moment of measurement rods:

$$M_{Mx}(\alpha Rx, \Delta zr) =$$

$$FMX1\left[\sin(\alpha_{Mx}(\alpha Rx, \Delta zr) \cdot L_{ry} + \cos(\alpha_{Mx}(\alpha Rx, \Delta zr)) \cdot (L_{rz}\cos(\alpha_{z}(\alpha Rx, \Delta yr)) + L_{bz})\right] + FMX2\left[\sin(\alpha_{Mx}(\alpha Rx, \Delta zr) \cdot L_{ry} + \cos(\alpha_{Mx}(\alpha Rx, \Delta zr)) \cdot (L_{rz}\cos(\alpha_{z}(\alpha Rx, \Delta yr)) + L_{bz} + 2L)\right]$$

Moment resulting from Fz and displacement ∆yr

 $M_{xFz}(\Delta yr) = F_z \cdot \Delta yr$ 

Moment resulting from Fy and displacement ∆yr

 $M_{xFy}(\alpha Rx, \Delta zr) = F_{y} \cdot L_{by} \cdot (\sin(\alpha_{y}(\alpha Rx, \Delta zr) - \alpha Rx))$ 

Total moments about the main lug center:

$$\sum Mx = 0 = -M_{xr} + M_{xFz} - M_{Mx} - F_y \cdot L_{by} \cdot \sin(\alpha_y (\Delta zr))$$

With the above mentioned equations the reaction moment Mxr is

$$M_{xr}(\alpha Rx, \Delta yr, \Delta zr) = -M_{xFz}(\Delta yr) - M_{Mx}(\alpha Rx, \Delta yr, \Delta zr) - M_{xFy}(\alpha Rx, \Delta zr)$$

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# 9.6.3 Local lug moment Mz (Equation considers displacements in xy-plane)

Test rig dimension:

Dimension			Description
L <sub>Fy</sub>	[mm]	1783	Y-cylinder length from outer to inner bearing point
L <sub>by</sub>	[mm]	720	Inner Y-cylinder bearing point to the lug reference point
L <sub>bx</sub>	[mm]	0	The bolt axis is aligned with the Y-cylinder axis
L <sub>xry</sub>	[mm]	300	Half distance between the z-axis main rods X1 and X2
L <sub>rx</sub>	[mm]	1990.6	Length of the main rods X1 and X2
L <sub>FMz</sub>	[mm]	2000	Length of the moment measurement rods FMZ1 and FMZ2
L <sub>x</sub>	[mm]	525	Half distance between FMZ1 and FMZ2

Equations to recalculate the local lug reactions with the measured data.

**Deformation angle:** 

$$\alpha_{y}(\Delta xr) = \arcsin\left(\frac{\Delta xr}{L_{Fy}}\right)$$
  $\alpha_{x}(\Delta yr) = \arcsin\left(\frac{\Delta yr}{L_{rx}}\right)$   $\alpha_{Mz}(\Delta xr) = \arcsin\left(\frac{\Delta xr}{L_{FMz}}\right)$ 

Total moment of measurement rods:

 $M_{Mz}(\Delta xr, \Delta yr) = (F_{Mz1} + F_{Mz2}) \cdot \left[ \sin(\alpha_{Mz}(\Delta xr)) \cdot L_{xry} + \cos(\alpha_{Mz}(\Delta xr)) \cdot (L_{rx} \cdot \cos(\alpha_x(\Delta yr)) + L_{bx}) \right] + \cos(\alpha M_z(\Delta xr)) \cdot F_{Mz1} \cdot 2 \cdot L_x$ 

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Moment resulting from Fx and displacement ∆yr

 $M_{zFx}(\Delta yr) = -F_x \cdot \Delta yr$ 

Moment resulting from Fy and displacement  $\Delta xr$ 

 $M_{zFy}(\Delta xr) = F_y \cdot \Delta xr \cdot (\cos(\alpha_y(\Delta xr)) + \sin(\alpha_y(\Delta xr)))$ 

With the above mentioned equations the reaction moment Mzr is

 $M_{zr}(\Delta xr, \Delta yr) = M_{zFx}(\Delta yr) + M_{Mz}(\Delta xr, \Delta yr) + M_{zFy}(\Delta xr)$ 

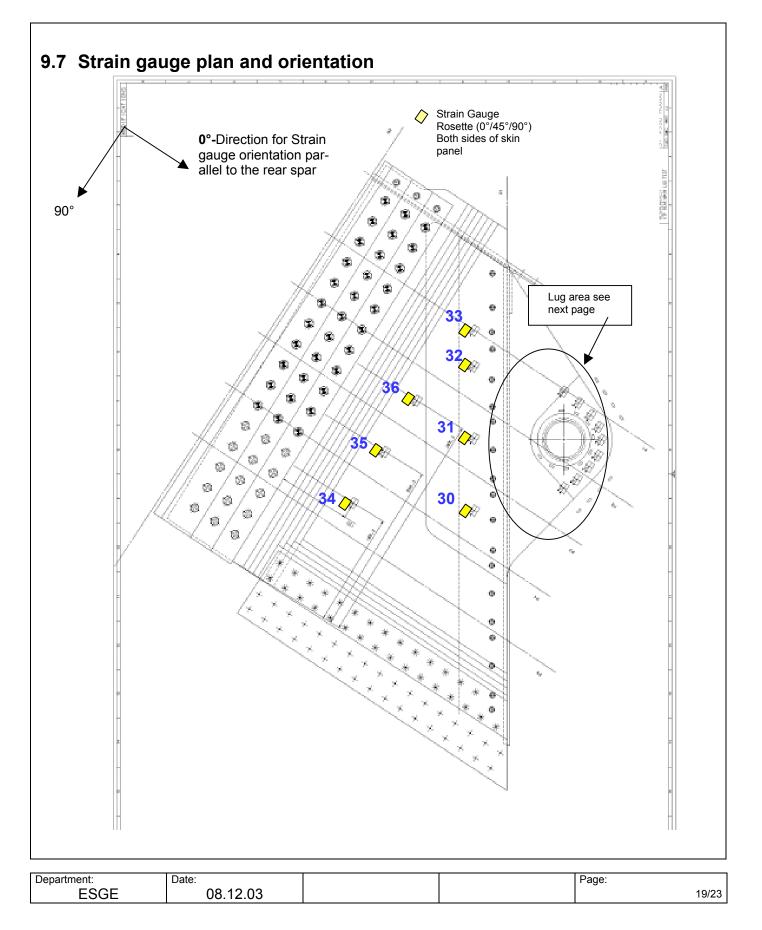
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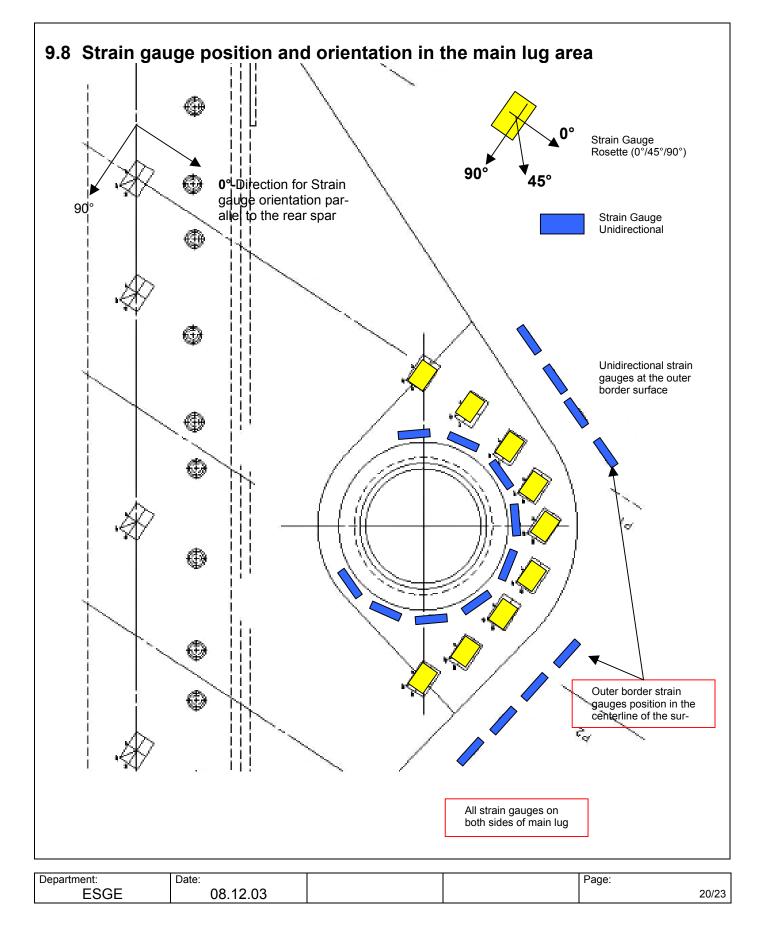
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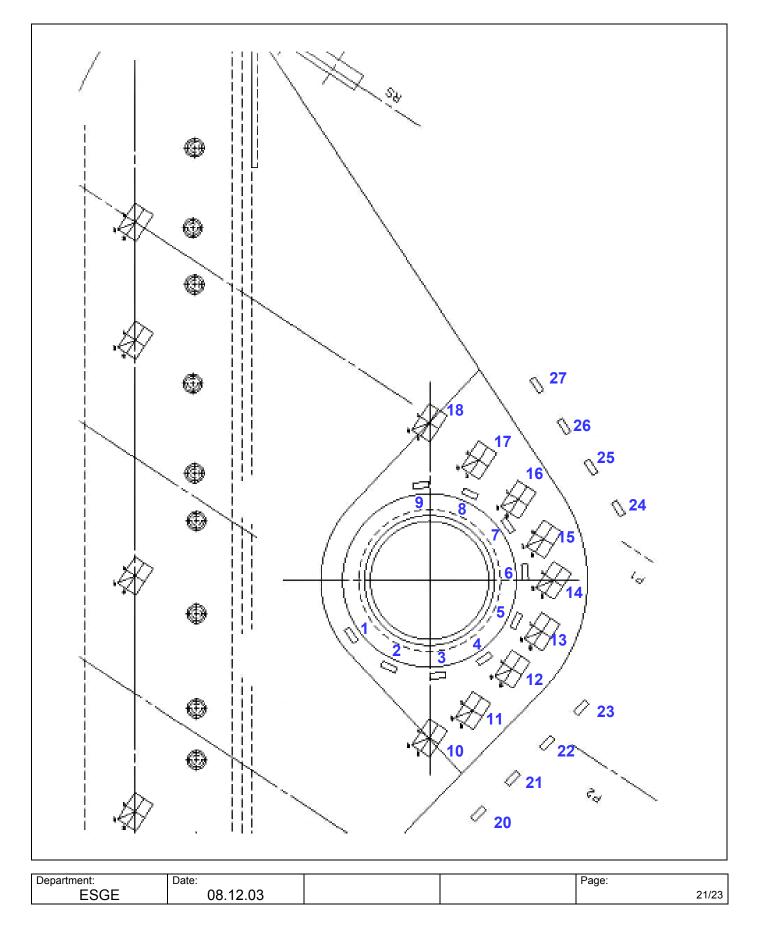




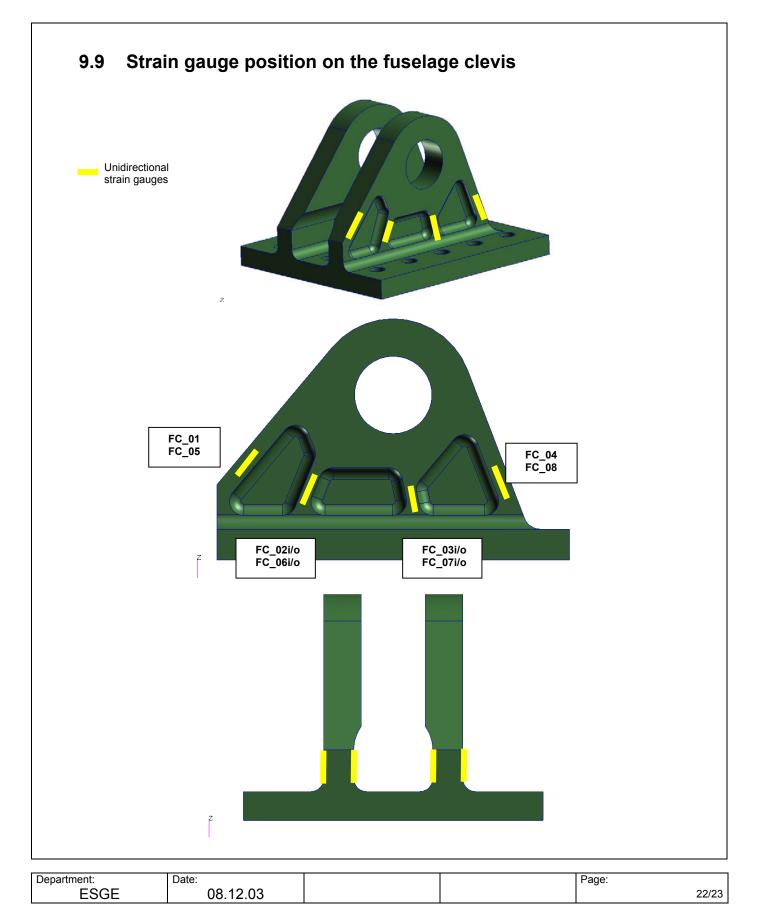
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